1 c 1 E International Centre for Indoor Environment and Energy **INODOOR BUILT ENVIRONMENT –** challenges for health, comfort and productivity versus energy use?

Professor Bjarne W. Olesen, Ph.D. www.ie.dtu.dk

Technical University of Denmark

DTU

Buildings use in developped countries 42 % of total energy consumption

- Heating
- Cooling
- Ventilation
- Domestic Hot Water
- Lighting
- Houshold appliances

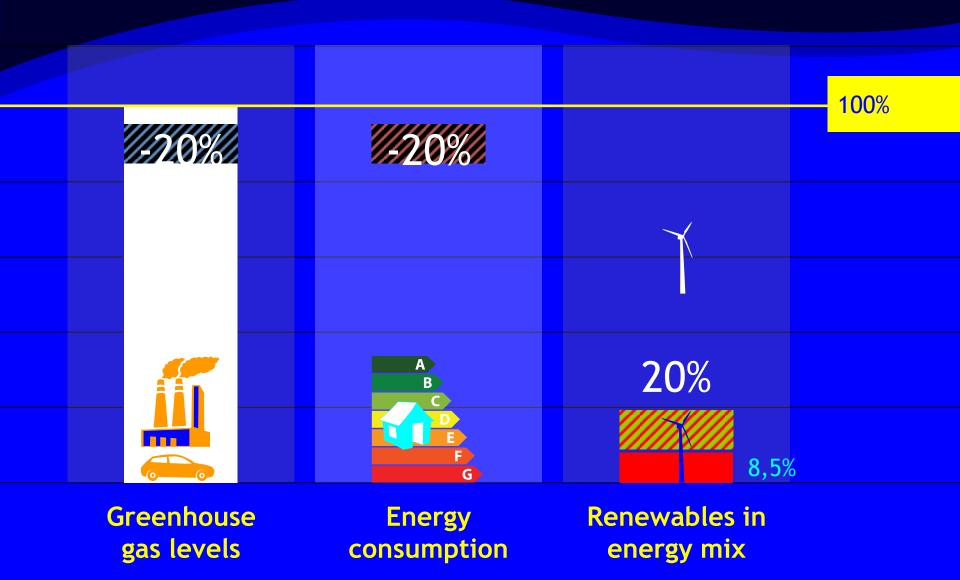
Energy Demand-Energy Efficiency-Energy Sources

- As the resources of fossil fuels are limited and the use of nuclear power is associated with several safety issues there is a worldwide need for reducing the energy demand of buildings.
- Reducing the energy demand is much more efficient and sustainable than increasing energy sources
- When constructing new buildings and renovation old buildings the first priority is to decrease the energy demand

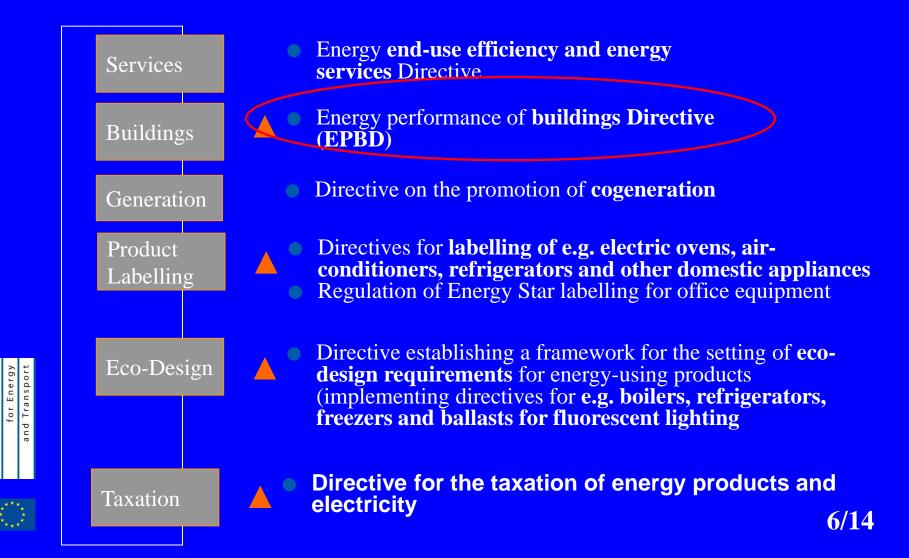
POSSIBLE ACTIONS

- Decrease energy demands (building design)
- Increase energy efficiency (HVAC systems)
- Increase use of renewable energy sources (wind, solar, geothermal, biomass)
- New energy sources (fuel cell)

The 20-20-20 EU policy by 2020

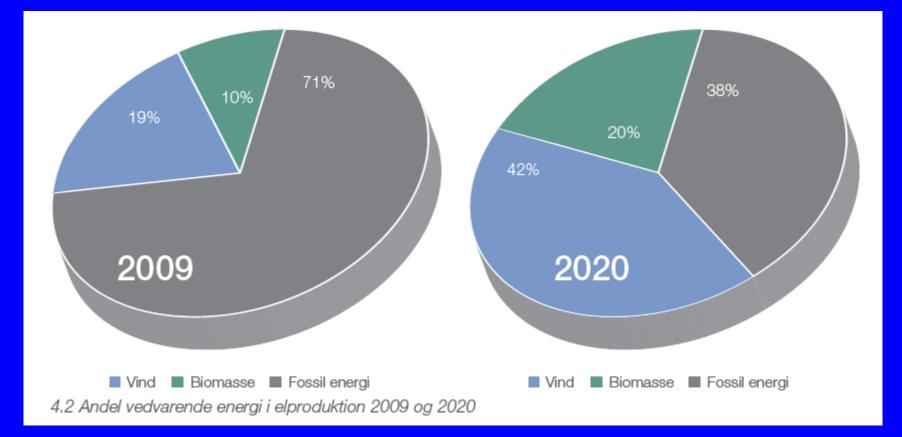


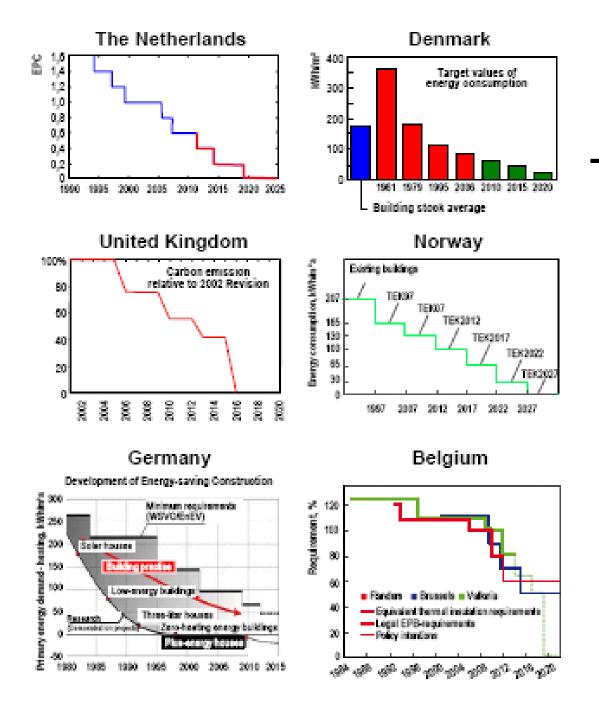
Comprehensive set of legislation to enhance energy efficiency



Directorate-General

Part of renewable energy sources (wind and bio-fuel) in Denmark (no nuclear)





The effect og building regulations

COMFORT-PRODUCTIVITY Building costs

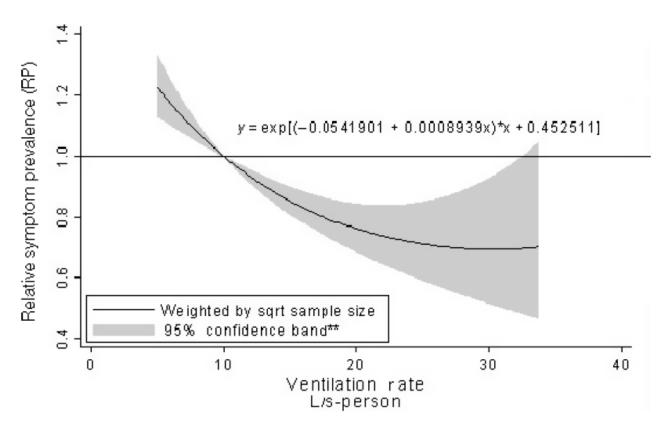
People 100

Maintenance 10

- Financing 10
- Energy 1

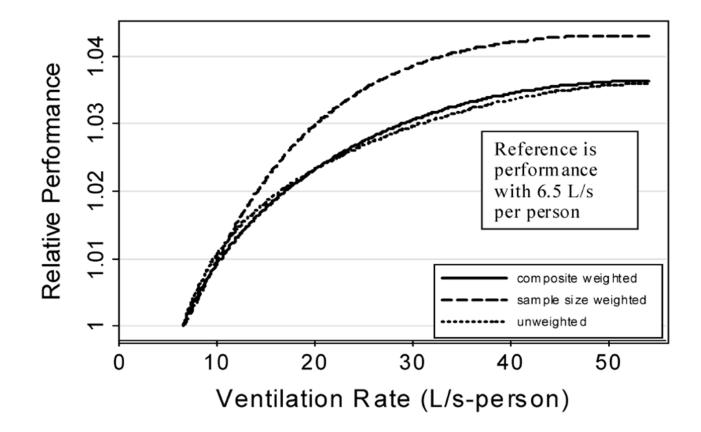
This clearly show that buildings are for people not for saving energy

Ventilation rates too low? SBS Symptoms



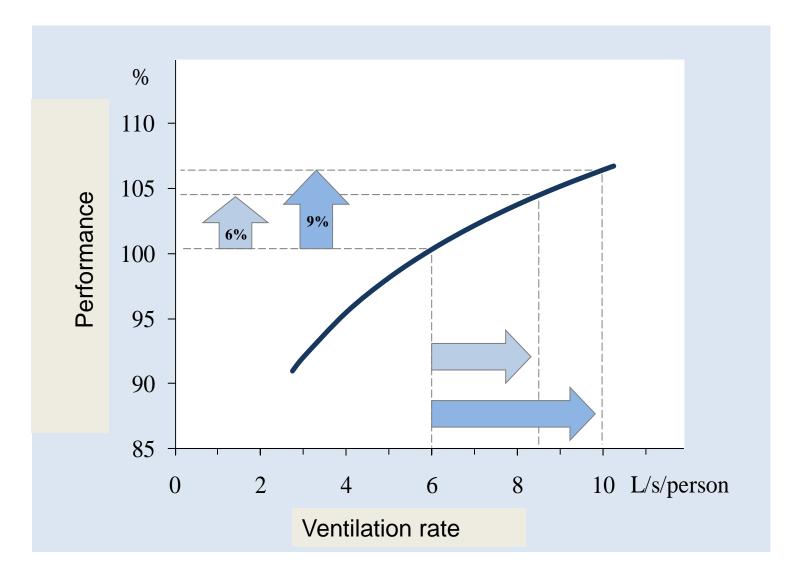
Source: W. Fisk, A Mirer, M. Mendell. 2009. Quantitative relationship of sick building syndrome symptoms with ventilation rates. Indoor Air

Ventilation rates too low? Productivity



Source: Seppänen, O. and W. Fisk. 2006. Some Quantitative Relations between Indoor Environmental Quality and Work Performance or Health. HVAC&R Research.

Ventilation and performance of school work





PMV-values -1.5 -1.0 -0.5 0 0.5 1.0 1.5 2.0 <u>_</u> .95 o_. .85 ø composite weighted sample size weighted unweighted 20 25 15 30 35

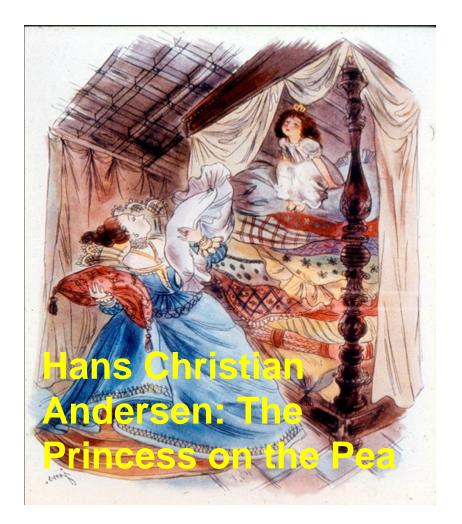
Temperature (C)

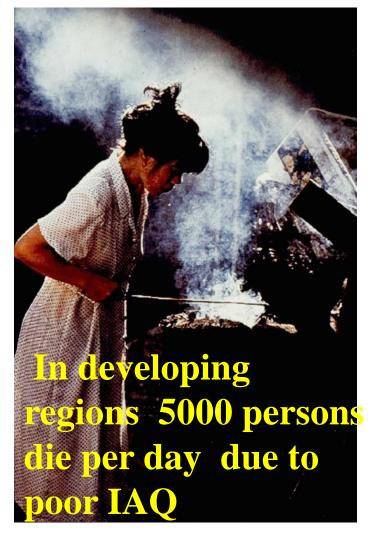


from Seppänen and Fisk 2005a

International Centre for Indoor Environment And Energy

Global impact on people





Achieving Excellence in Indoor Environmental Quality

- Physical factors
 - Thermal Comfort
 - Air quality (ventilation)
 - Noise-Acoustic
 - Illumination
- Personal factors
 - Activity
 - Clothing
 - Adaptation
 - Expectation
 - Exposure time

STANDARDS

• ISO EN 7730-2005

- Ergonomics of the thermal environment Analytical determination and interpretation of thermal comfort using calculation of the PMV and PPD indices and local thermal comfort effects.
- ASHRAE 55-2013
 - Thermal environment conditions for human occupancy
- ASHRAE 62.1 and 62.2 -2013
 - Ventilation and indoor air quality
- EN15251
 - Indoor environmental input parameters for design and assessment of energy performance of buildings- addressing indoor air quality, thermal environment, lighting and acoustic
- EN 13779
 - Ventilation for non-residential buildings performance requirements for ventilation and room-conditioning systems

International Standards Indoor Environmental Quality

- prEN 16789-1 (revision EN15251) and ISO CD 17772
 - Indoor environmental input parameters for the design and assessment of energy performance of buildings.
- DTR 16789-2 and ISO NWI TR 17772:
 - Guideline for using indoor environmental input parameters for the design and assessment of energy performance of buildings.

Categories

Cate- gory	Explanation		
Ι	High level of expectation and also recommended for spaces occupied by very sensitive and fragile persons with special requirements like some disabilities, sick, very young children and elderly persons, to increase accessibility.		
II	Normal level of expectation		
- 111	An acceptable, moderate level of expectation		
IV	Low level of expectation. This category should only be accepted for a limited part of the year		

MODERATE THERMAL ENVIRONMENTS

- GENERAL THERMAL COMFORT

 PMV / PPD, OPERATIVE TEMPERATURE
- LOCAL THERMAL DISCOMFORT
 - Radiant temperature asymmetry
 - Draught
 - Vertical air temperature difference
 - Floor surface temperature

Recommended categories for design of mechanical heated and cooled buildings

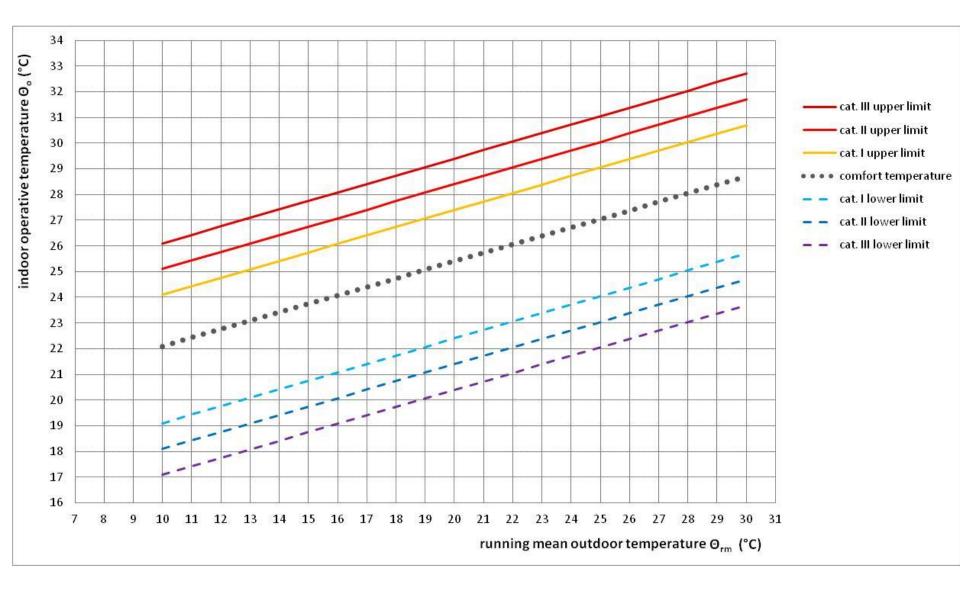
Category	Thermal state of the body as a whole		
	PPD %	Predicted Mean Vote	
I	< 6	-0.2 < PMV < + 0.2	
II	< 10	-0.5 < PMV < + 0.5	
ш	< 15	-0.7 < PMV < + 0.7	
III	< 25	-1.0 < PMV < + 1.0	

Temperature ranges for hourly calculation of cooling and heating energy in three categories of indoor environment

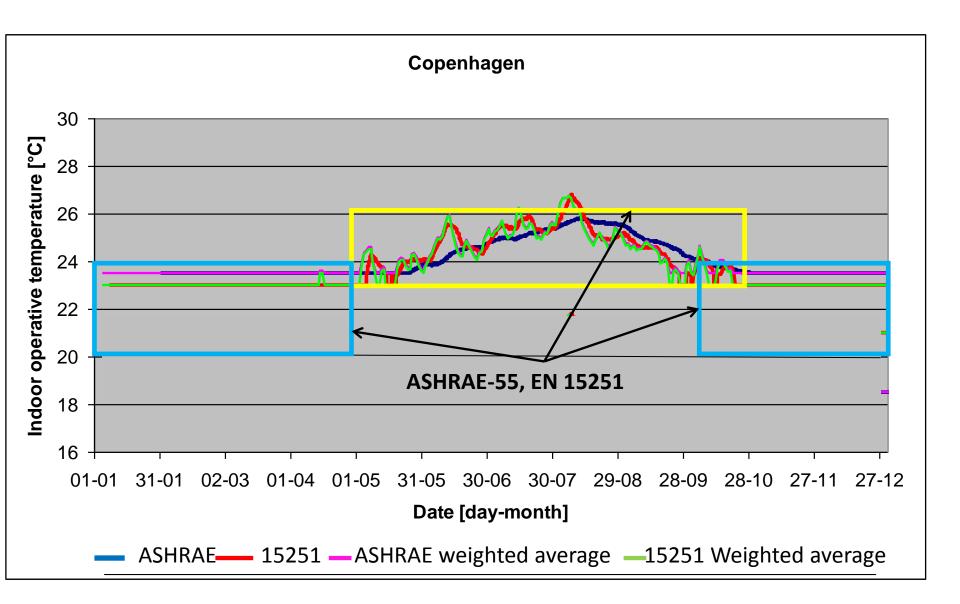
Type of building/ space	Category	Operative Temperature for Energy Calculations °C	
Offices and spaces with similar activity (single		Heating (winter season), ~ 1,0 clo	Cooling (summer season), ~ 0,5 clo
offices, open plan offices, conference rooms,	Ι	21,0 - 23,0	23,5 - 25,5
auditorium, cafeteria, restaurants, class rooms, Sadantary activity 1.2 mat	II	20,0 – 24,0	23,0 - 26,0
Sedentary activity ~1,2 met	III	19,0 – 25,0	22,0 - 27,0
	IV	17,0 – 26,0	21,0 - 28,0

ADAPTATION IN NATURAL VENTILATED BUILDINGS ?

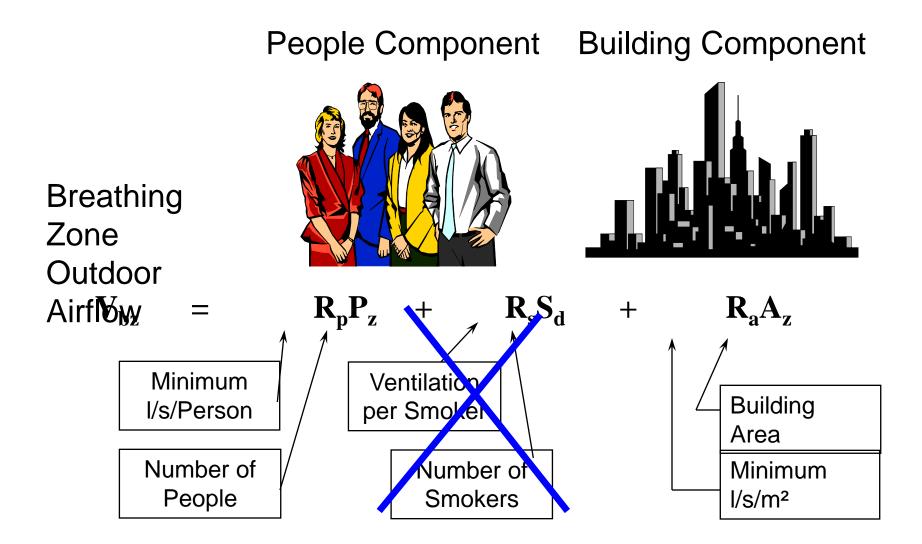
- Behavioural
 - Clothing, activity, posture
- Psychological
 - Expectations



 $\Theta_{\rm rm} = (\Theta_{\rm ed -1} + 0.8 \ \Theta_{\rm ed -2} + 0.6 \ \Theta_{\rm ed -3} + 0.5 \ \Theta_{\rm ed -4} + 0.4 \ \Theta_{\rm ed -5} + 0.3 \ \Theta_{\rm ed -6} + 0.2 \ \Theta_{\rm ed -7})/3.8$



Concept for calculation of design ventilation rate



Total ventilation rate $q_{tot} = n \cdot q_p + A_R \cdot q_B$ $q_{supply} = q_{tot} / \varepsilon_y$

- Where
- ε_v = the ventilation effectiveness (EN13779)
- q_{supply} = ventilation rate supplied by the ventilation system
- q_{tot} = total ventilation rate for the breathing zone, I/s
- *n* = design value for the number of the persons in the room,
- q_p = ventilation rate for occupancy per person, l/s, pers
- $A_R^{'}$ = room floor area, m²
- q_B = ventilation rate for emissions from building, l/s,m²

HEALTH CRITERIA FOR VENTILATION

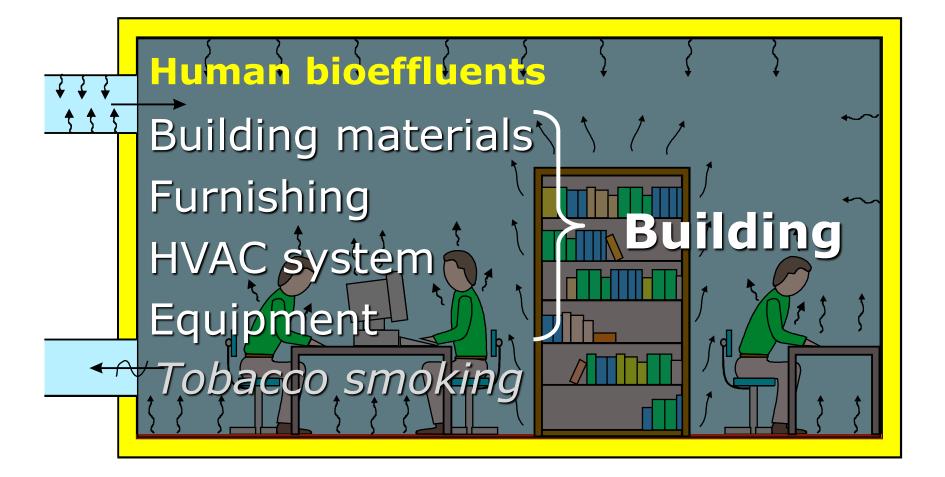
Minimum 4 l/s/person

Basic required ventilation rates for diluting emissions (bio effluents) from people for different categories

Category	Expected Percentage Dissatisfied	Airflow per non- adapted person I/(s.pers)
Ι	15	10
11	20	7
111	30	4
IV	40	2,5*

*The total ventilation rate must never be lower than 4 l/s per person ASHRAE Standard 62.1 : Adapted persons 2,5 l/s person (Cat. II)

Indoor pollution sources



Design ventilation rates for diluting emissions from buildings

Category	Very low polluting building I/(s m ²)	Low polluting building I/(s m ²)	Non low- polluting building I/(s m ²)
I	0,5	1,0	2,0
II	0,35	0,7	1,4
111	0,2	0,4	0,8
IV	0,15	0,3	0,6
Minimum total ventilation rate for health	4 I/s person	4 I/s person	4 I/s person

Example on how to define low and very low polluting buildings

SOURCE	Low emitting products for low polluted buildings	Very low emitting products for very low polluted buildings
Total VOCs TVOC (as in CEN/TS 16516)	< 1.000 µg/m³	< 300 µg/m³
Formaldehyde	< 100 µg/m³	< 30 µg/m³
Any C1A or C1B classified carcinogenic VOC	< 5 µg/m³	< 5 µg/m³
R value (as in CEN/TS16516)	< 1.0	< 1.0

Example of design ventilation air flow rates for a single-person office of 10 m² in a low polluting building (un-adapted person)

Cate- gory	Low- polluting building I/(s*m ²)	Airflow per non- adapted person I/(s*person)		design ventilation or the room I/(s*person)	n air flow I/(s* m²)
l	1,0	10	20	20	2
II	0,7	7	14	14	1,4
111	0,4	4	8	8	0,8
IV	0,3	2,5	5,5	5,5	0,55

Specific Pollutans

The ventilation rate required to dilute a pollutant shall be calculated by this equation:

$$Q_{h} = \frac{G_{h}}{C_{h,i} - C_{h,o}} \frac{1}{\epsilon_{v}} Eq (2)$$

where:

Q_h is the ventilation rate required for dilution, in litre per second;

G_h is the pollution load of a pollutant, in micrograms per second;

 $C_{h,i}$ is the guideline value of a pollutant, see Annex B6, in micrograms per m³;

 $C_{h,o}$ is the supply concentration of pollutants at the air intake, in micrograms per m³;

 ϵ_v is the ventilation effectiveness

NOTE. $C_{h,i}$ and $C_{h,o}$ may also be expressed as ppm (vol/vol). In this case the pollution load G_h has to be expressed as l/s.

Pollutant	WHO Indoor Air Quality guidelines 2010	WHO Air Quality guidelines 2005
Benzene	No safe level can be determined	-
Carbon monoxide	15 min. mean: 100 mg/m ³ 1h mean: 35 mg/m ³ 8h mean: 10 mg/m ³ 24h mean: 7 mg/m ³	-
Formaldehyde	30 min. mean: 100 µg/m³	-
Naphthalene	Annual mean: 10 µg/m ³	-
Nitrogen dioxide	1h mean: 200 μg/m ³ Annual mean: 40 mg/m ³	-
Polyaromatic Hydrocarbons (e.g. Benzo Pyrene A B[a]P)	No safe level can be determined	-
Radon	100 Bq/m ³ (sometimes 300 mg/m ³ , country-specific)	-
Trichlorethylene	No safe level can be determined	-
Tetrachloroethylene	Annual mean: 250 µg/m³	
Sulfure dioxide	-	10 min. mean: 500 μg/m³ 24h mean: 20 mg/m³
Ozone	-	8h mean:100 µg/m³
Particulate Matter PM 2,5	-	24h mean: 25 μg/m ³ Annual mean: 10 μg/m ³
Particulate Matter PM 10	-	24h mean: 50 μg/m ³ Annual mean: 20 μg/m ³

WHO guidelines values for indoor and outdoor air pollutants

Energy Efficient Technologies

- Indoor air quality
 - Reduce loads (pollution sources)
 - Heat recovery
 - Increase system efficiency
 - Natural ventilation-Hybrid ventilation
 - Air distribution (contaminant removal) effectiveness
 - Personal ventilation
 - Air cleaning
- Thermal comfort
 - Reduce loads (building shell, solar screen, internal loads)
 - Increase system efficiency
 - Low Temperatur Heating- and High Temperature Cooling Systems
 - Use of building mass to reduce peaks (Thermo-Active-Building-Systems (TABS))
 - Drifting indoor temperatures

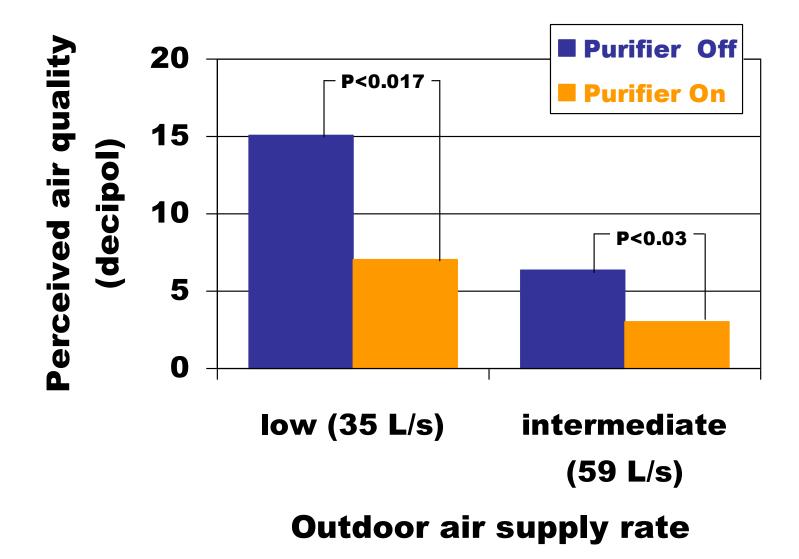
Trends regarding ventilation

- Increasing use of air cleaning
 - Filtration
 - Gas phase air cleaning
- Personalized systems for better comfort and energy savings
- Demand control ventilation
 - Occupant precense
 - $-CO_2$ sensors
 - Artificial nose

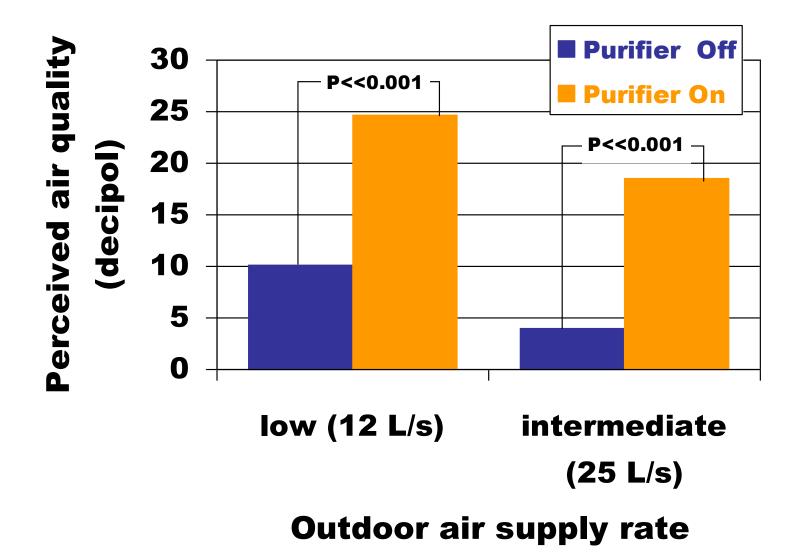
Gas phase air purification technologies

- Photo-catalytic oxidation (PCO)
- Ozone oxidation
- Thermal catalytic oxidation (TCO)
- Plasma oxidation
- Botanic filtration
- Sorption filtration

Results: Bldg mat, PCs, filters

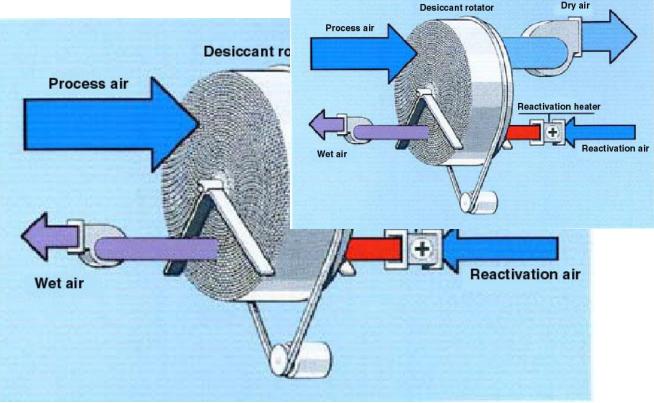


Results: Human bio effluents

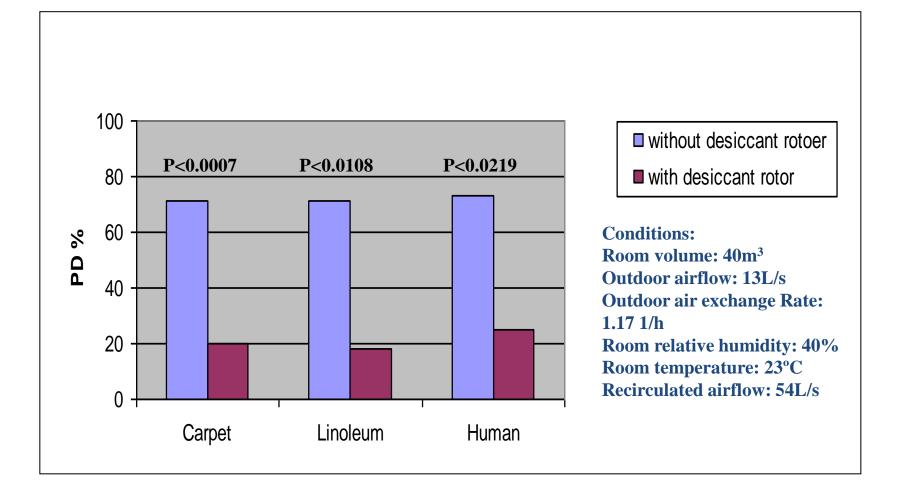


How can a desiccant rotor remove pollutants?

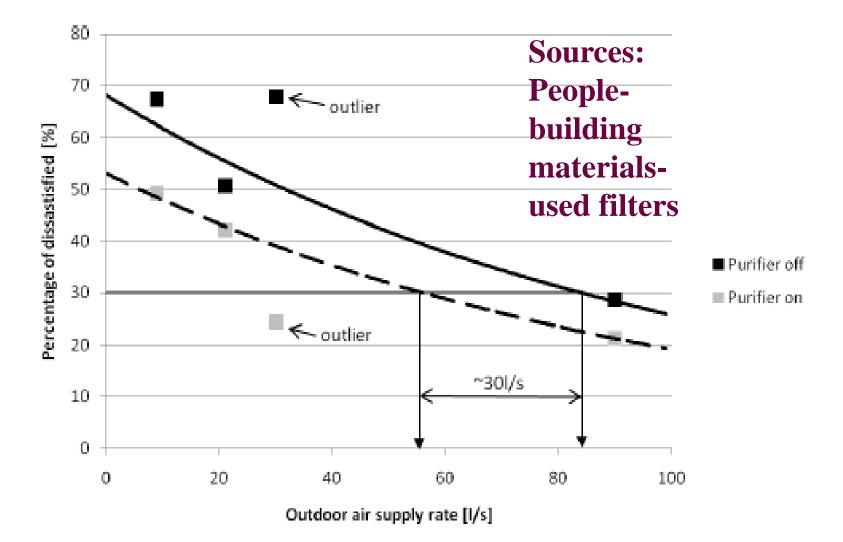
- Adsorptio
 chemicals
 the procession
 the rotor
- Desorptio
 from the (reactivation
 the rotor



PAQ in PD% with and without desiccant rotor



Effect of air cleaning on perceived Air Quality



Trends regarding ventilation

- Increasing use of air cleaning
 - Filtration
 - Gas phase air cleaning
- Personalized systems for better comfort and energy savings
- Demand control ventilation
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 - Artificial nose

Air Distribution Effectiveness

$$\varepsilon_{V} = \frac{C_{E} - C_{S}}{C_{I} - C_{S}}$$

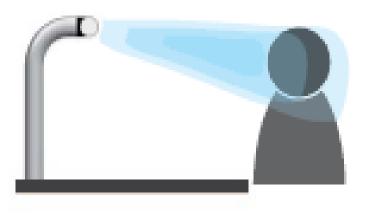
Concentrations:

- C_E exhaust air
- C_s supply air
- C₁ breathing zone

CEN Report CR 1752 (1998)

Mixing ventilation		Mixing ventilation		Displacement ventilation		Personalized ventilation	
T supply -	Vent. effect.	T supply -	Vent. effect.	T supply -	Vent. effect.	T supply -	Vent. effect.
T inhal		T inhal		T inhal		T room	
°C	-	°C	-	°C	-	°C	-
< 0	0,9 - 1,0	< -5	0,9	<0	1,2 - 1,4	-6	1,2 - 2,2
0 - 2	0,9	-5 - 0	0,9 - 1,0	0-2	0,7 - 0,9	-3	1,3 - 2,3
2 - 5	0,8	> 0	1	>2	0,2 - 0,7	0	1,6 - 3,5
> 5	0,4 - 0,7						

Personalized systems





- Can "standard" criteria be directly applied also to a personal system, where the occupants try to meet their own preferences?
- If the occupants have a personalized system is it then possible to relax on the requirements to the general environment?
- Increased satisfaction

Personalized Ventilation





Personalized Ventilation in Offices









Energy Efficient Technologies

- Indoor air quality
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 - Drifting indoor temperatures

Low-Temperature heating High-Temperature Cooling

- Heat exchange through large surfaces (floor, ceiling, walls)
- Supply water temperatures:
 - Heating: 25 40 °C
 - Cooling: 16 23 °C (temperature limited by dewpoint to avoid condensation)
- Wide range of systems, solutions both for residential and non-residential buildings

COMBINATION WITH LOW ENERGY SOURCES Day

- Heating supply temp. : 25 40°C
 - heat pumps
 - condensing boiler
 - ground coupling
 - waste heat
 - solar energy
- Cooling supply temp. : 16 23°C
 - reversible heat pump
 - ground coupling
 - free cooling
 - air cooled chillers





UPONOR Corporation (2010)

Cooling method



Ground water



Geothermal heat/coolth



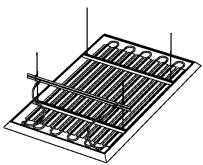
Night air

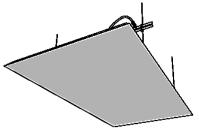


Cooling unit

Cooling panels





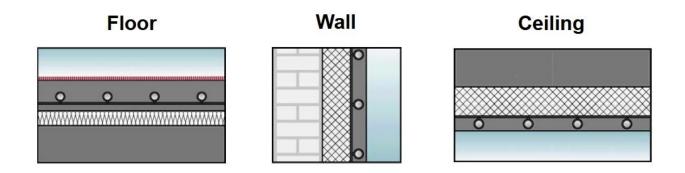


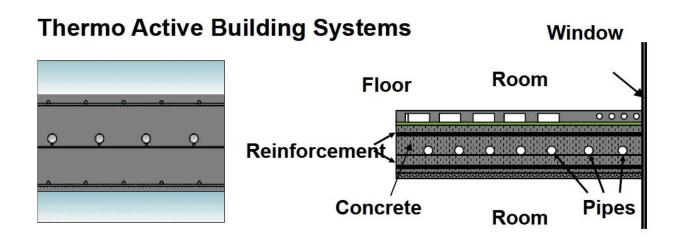




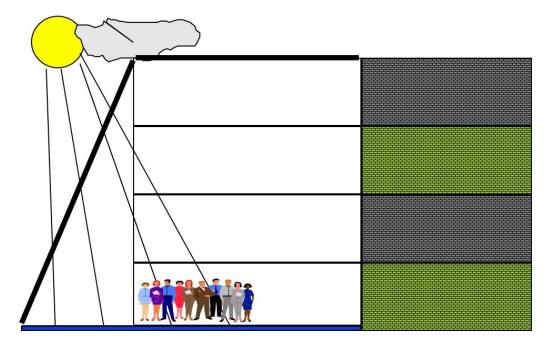
System types

- Water as the heat carrier
- Heat exchange is > 50% radiant
- Different installation concepts (thermally coupled or insulated form the building structure)





Additional benefits – large atriums and foyers



- The under-floor cooling system directly removes solar heat gains
- Minimum of such gains influences air temperature
- Comfortable floor surface temperature is maintained at the same time

Large atriums and foyers - examples Copenhagen Opera

House

Bangkok Airport

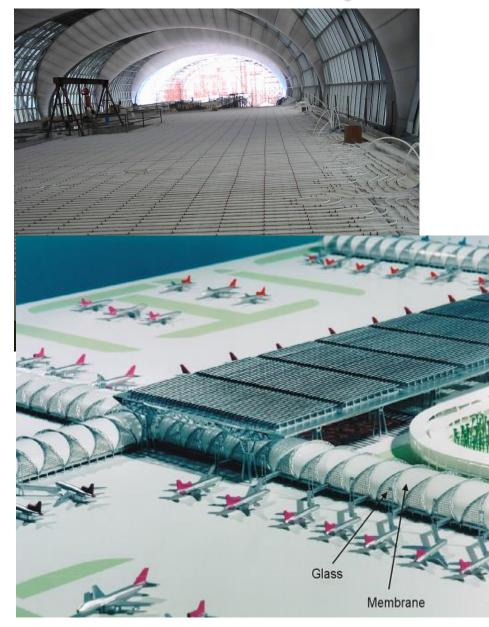


Opera House in Copenhagen: Cooling

- 2,5 MW cooling capacity
- 2 systems 10/15°C and 15/18 °C
- Free cooling from sea water
- Combined radiant floor heating + radiant floor cooling
- 18 km underfloor cooling tubing
- Quiet cooling walls
- De-humidification condensing coils for ventilation

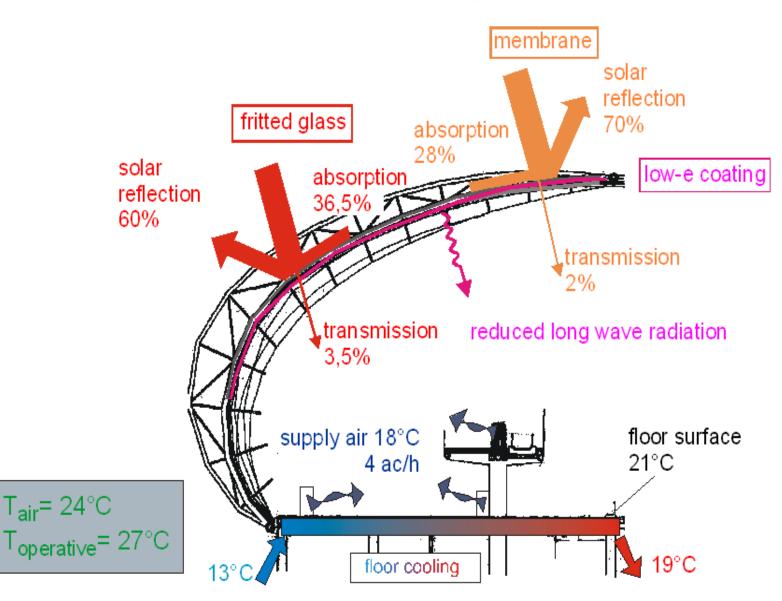


Airport Bangkok

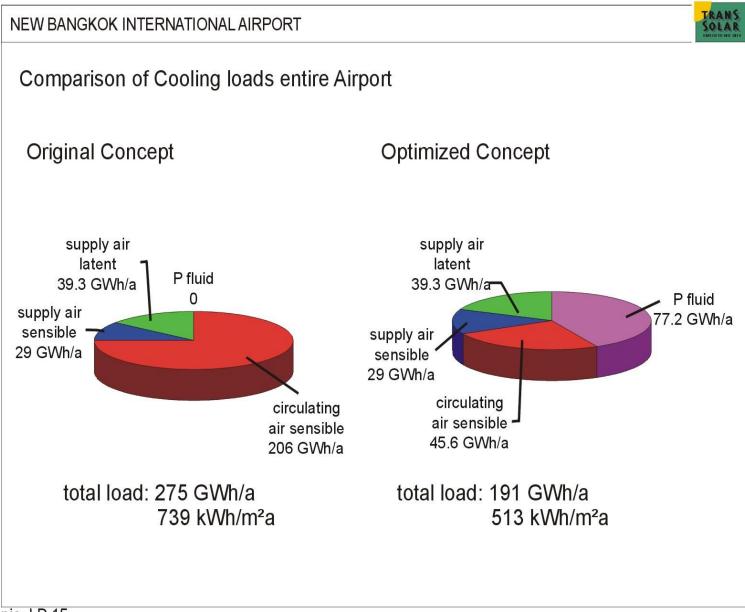




Airport Bangkok



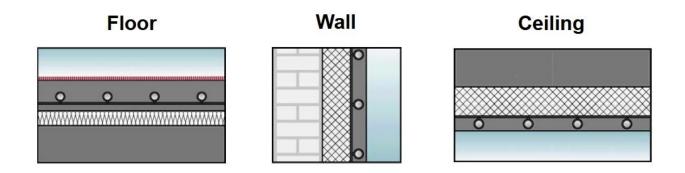
Airport Bangkok

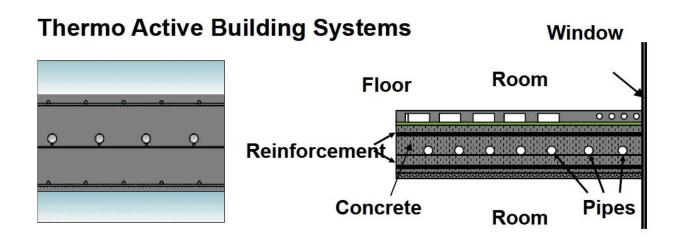


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System types

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Concept of Thermo Active Building Systems

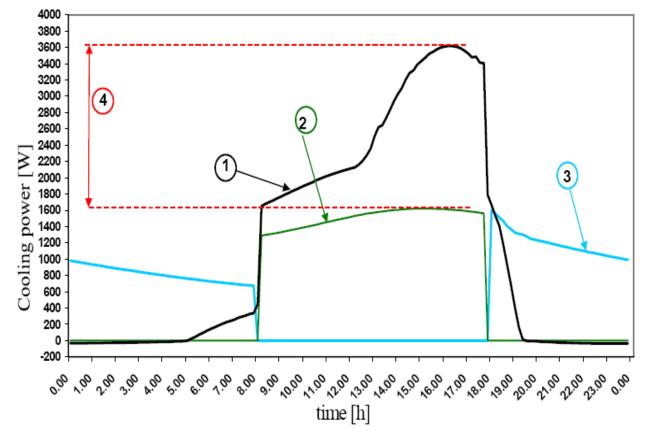
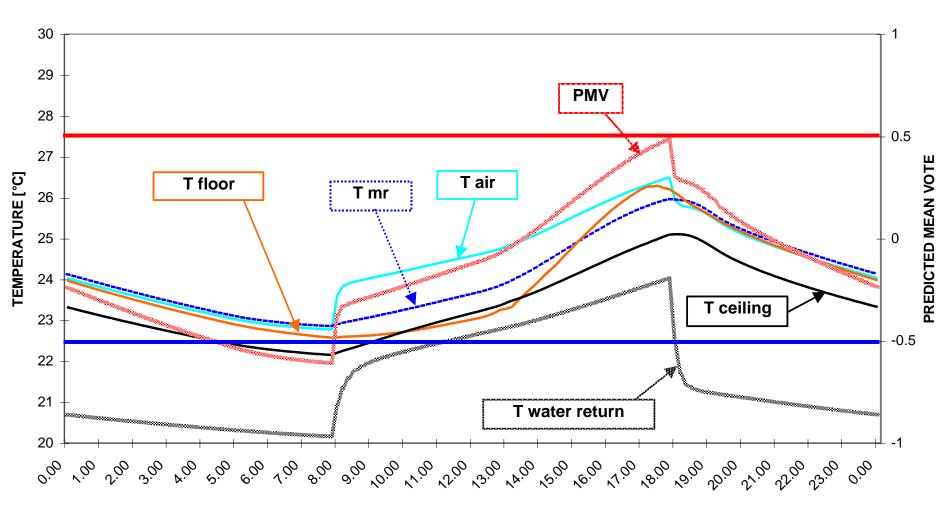


Figure 2 – Example of peak-shaving (reducing the peak load) effect (time vs. cooling power [W],)

Where: 1) heat gain, 2) Power needed for conditioning the ventilation air, 3) Power needed on the water side, 4) Peak heat gain reduction.

Concept of Thermo Active Building Systems (TABS)

EXAMPLE OF INTERNAL CONDITIONS WITH THERMAL SLAB

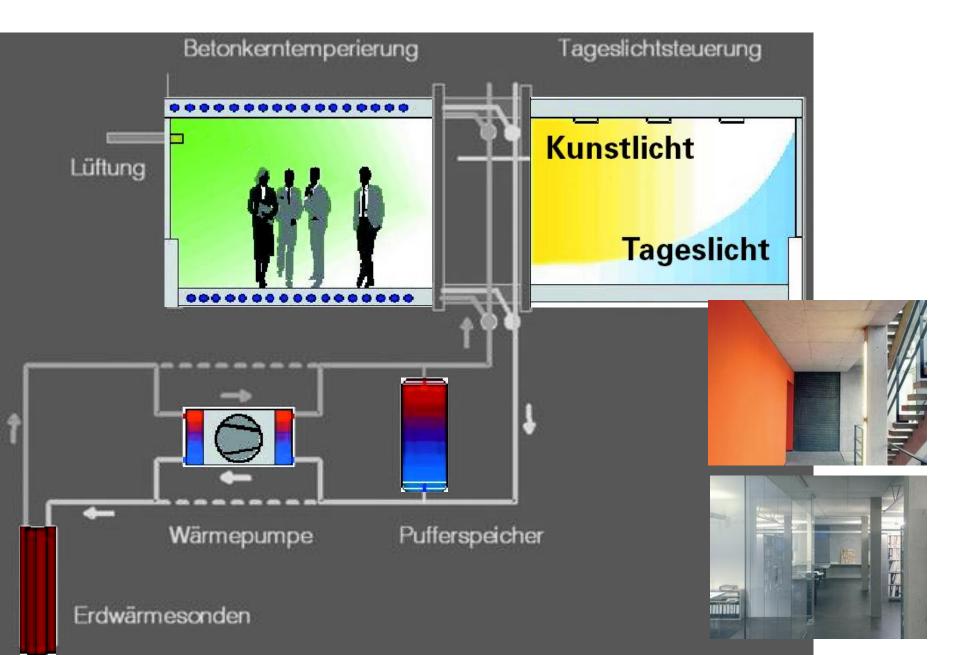


Balanced Office Building (BOB.1) Aachen, Germany

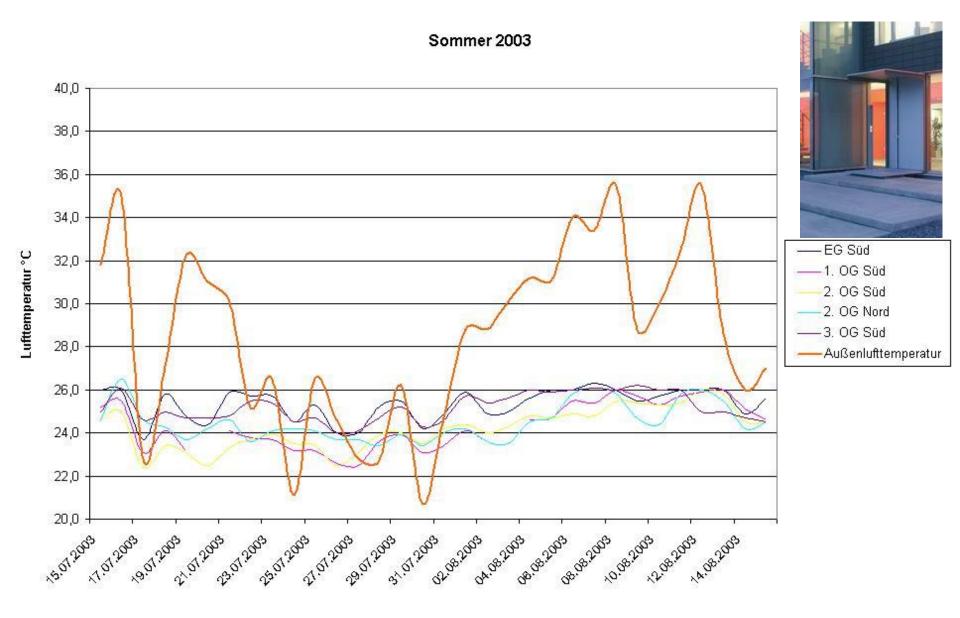
- Gross floor area 2,151 m²
- 4 storeys
- Efficiently insulated external envelope
- Ground-coupled heating and cooling with TABS
- Ventilation system with heat recovery
- Daylight-controlled lighting
- Rainwater collection for use in toilet flushing



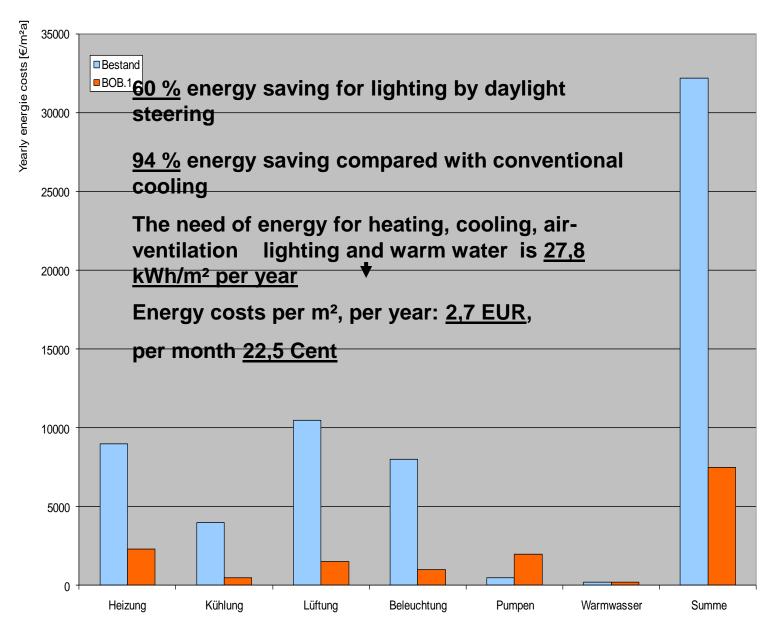
Energy concept in BOB.1



cooling period in BOB.1



Energy efficiency of BOB.1



THE WORLD'S LARGEST SIDE BY SIDE COMPARISON OF VAV AND RADIANT COOLING



Figure 1 - Infosys SDB-1 Hyderabad - 125,000 sf of radiant cooling and 125,000 sf of VAV cooling

Sun shading and daylight penetration RADIANT VAV

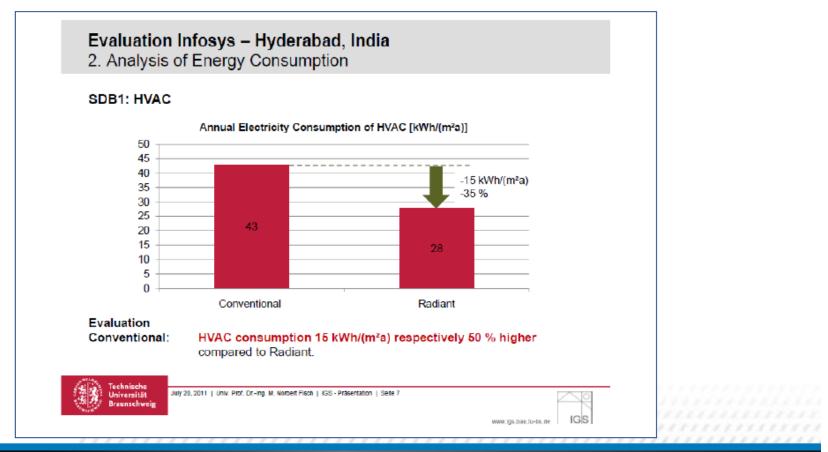


Infosys* It iicoyo

Building Tomorrow's Enterprise

Energy

Radiant Cooling – Third Party Evaluation



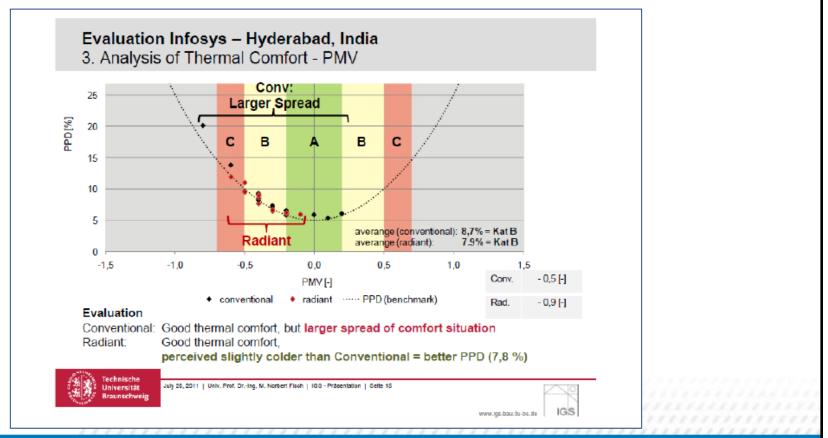
Building

Tomorrow's Enterprise

Infosys*

Thermal Comfort

Radiant Cooling - Third Party Evaluation



Building

Tomorrow's Enterprise

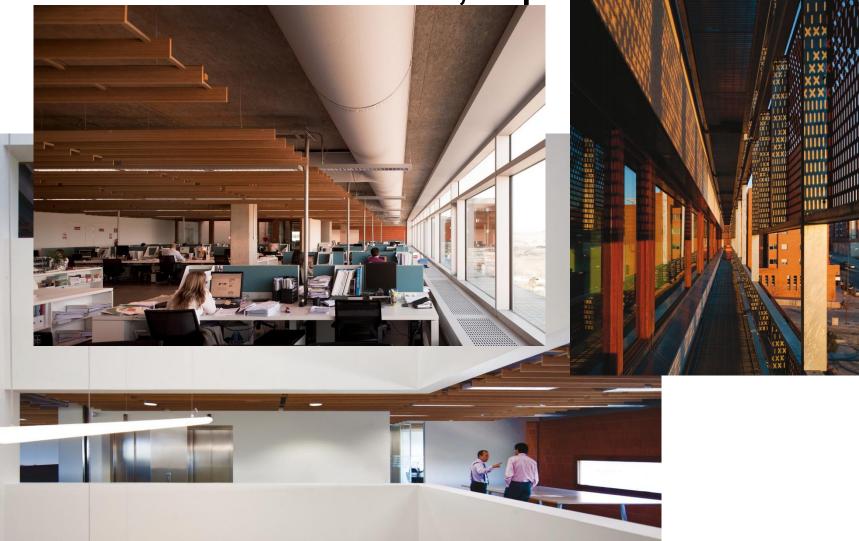


Radiant Cooling – Advantages

- Air system is about 1/5th of a conventional air conditioned building lesser ducting and lower fan power
- Water is the main medium of heat transfer pumping energy much smaller compared to fan energy.
- Chilled water temperature in the radiant pipes is 15 to 18 °C chillers run at high efficiency
- Perception of thermal comfort is higher compared to a conventional air conditioned building



IDOM Company Headquaters, Madrid, Spa



IDOM Company Headquaters, Madrid, Spain

- 16 000 m2
- Natural & Mechanical ventilation
- External solar shading & green facade
- TABS combinned with free cooling (covers 40-50 kWh/m2)



Energy use

(kWh/m²y)	IDOM HQ	CTE - MADRID	%
Heating + DHW	27,35	77,00	-64,5
Cooling	12,58	85,00	-85,2
Lighting	11,37	34,00	-66,6
Total	51,30	196,00	-73,8

IDOM Company Headquaters, Madrid, Spain

Primary energy in building services (kWh/m²y)

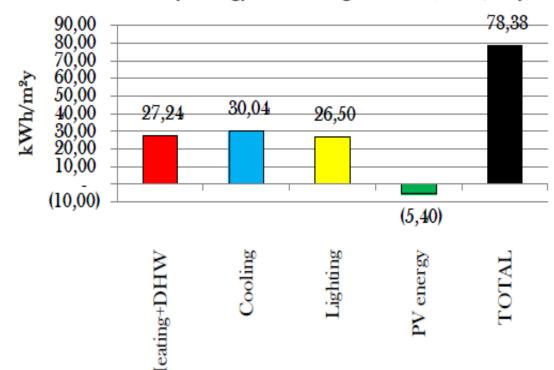


Figure 38. Primary Energy for building services in IDOM Headquarter.

General information



General view of industrial area in Padova

- Context, dimensions, costs
- 2.200 mq
- 8.000 mc
- 4 floors (1undergr.)
- 750 mq open

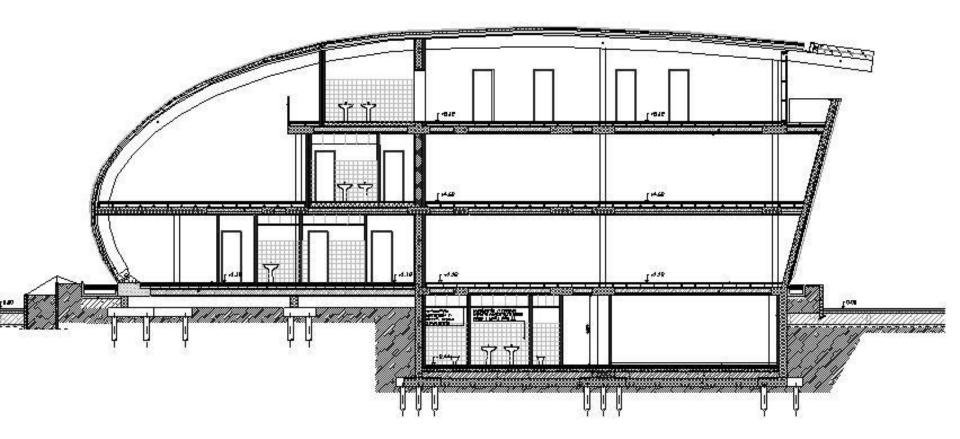
The building

Construction costs

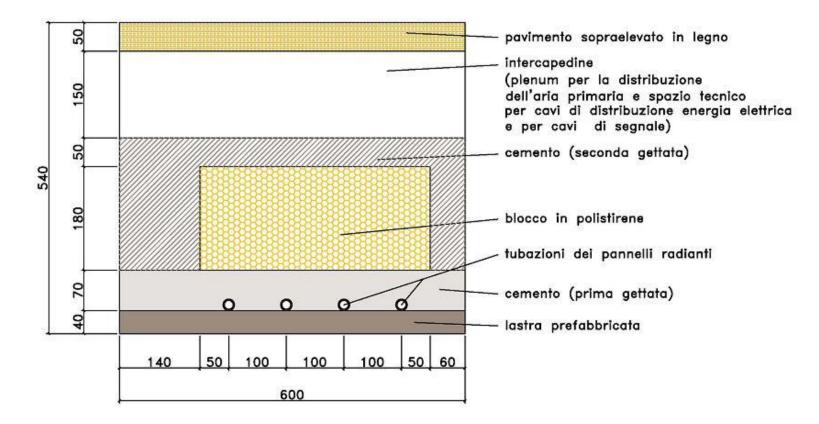
- •1.250 euro per sq.m
- 344 euro per cu.m



Longitudinal section of the OFFICE BUILDING



Active thermal slab radiant panel

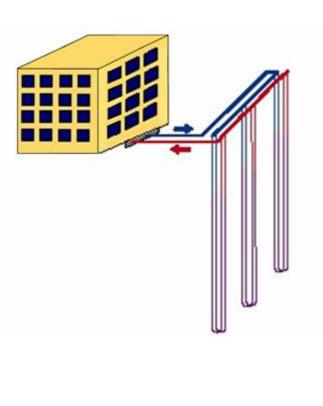


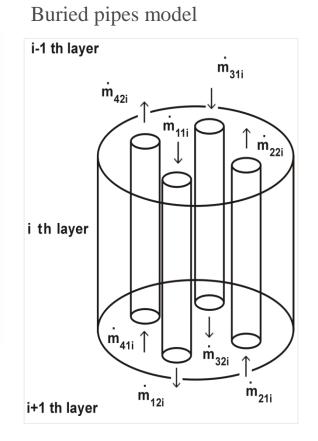
ATTIVAZIONE TERMICA DELLA MASSA

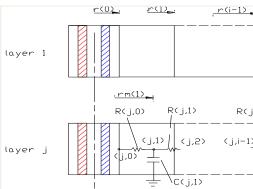
Heat pump and ground coupled heat exchanger

The ground heat exchanger

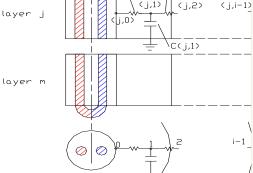
Sketch







Ground models



R(j

Installing the Ground Heat Exchangers















HEATING PERIOD

129 days

COP average~ 5,7

Specific primary energy requirement : 13,7 kWh_p/($m^2 Y$)

Specific heating requirement : $19 \text{ kWh}_t/(\text{m}^2 \text{ Y})$

Specific electrical energy requirement : 4,9 kWh_e/(m² Y)

COOLING PERIOD

175 days

SEER ~ 5,4

Specific primary energy requirement: 21,4 kWh_p/(m² Y)

Specific cooling requirement: 29,1 kWh_f/(m^2 Y)

Specific electrical energy requirement: 7,7 kWh_e/(m² Y)



Occupant behaviour, Indoor environment and energy consumption

 $f(x + \Delta x) = \sum_{i=0}^{\infty} \frac{(\Delta x)^{i}}{i!}$

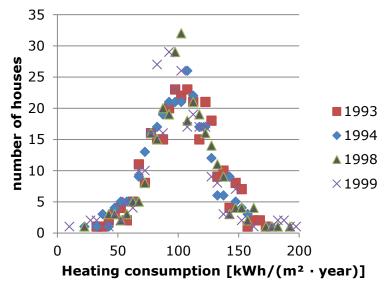
International Centre for Indoor Environment and Energy

DTU Civil Engineering Department of Civil Engineering

Investigation of heat consumption in 290 identical houses*

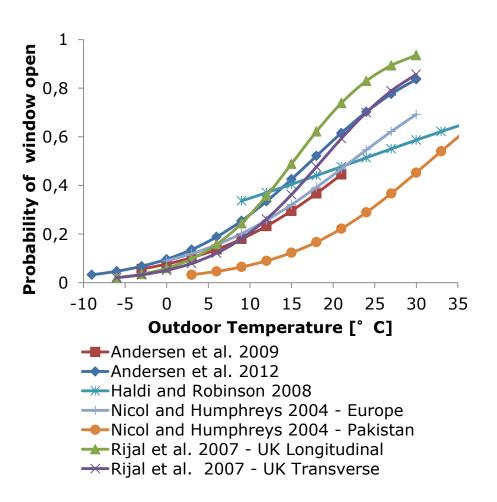
- Correction for differences in outer wall area
 - -End houses vs. Middle houses
- Highest consumption up to 20 times higher than lowest
- Stable consumption distribution over time
- No measurements of indoor environment





Models of occupants' window opening behaviour

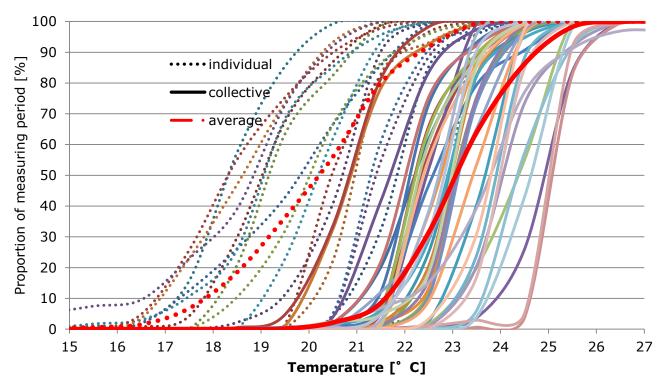
- Many models
- Most, only rely on thermal environment
 - -Is that enough?
 - -Which one should I use?
- Lack of validation
- Lack of validation methods



Shared or individual heating cost

Interviews of 10 residents

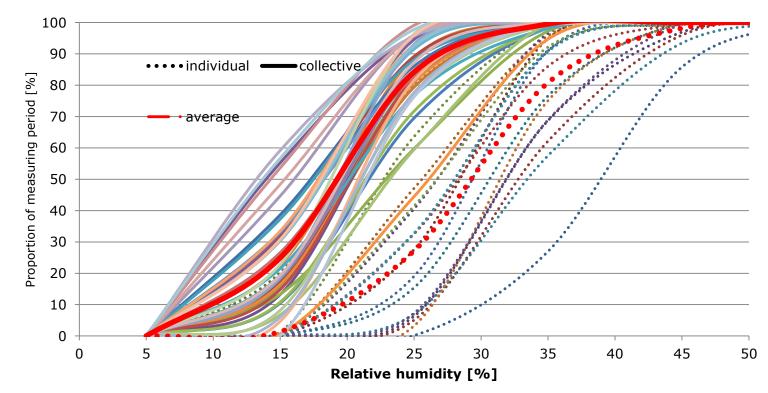
- Possible to heat all apartments to comfortable condition
- Individual billing
 - focus on heat savings
 - Accepted uncomfortable conditions to save money
- Collective billing
 - Focus on health, comfort and avoiding moisture problems



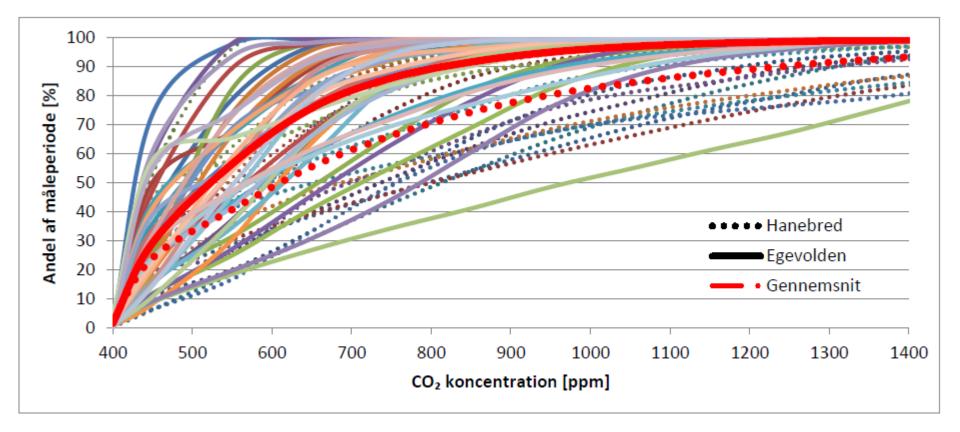
Shared or individual heating cost

Interviews of 10 residents

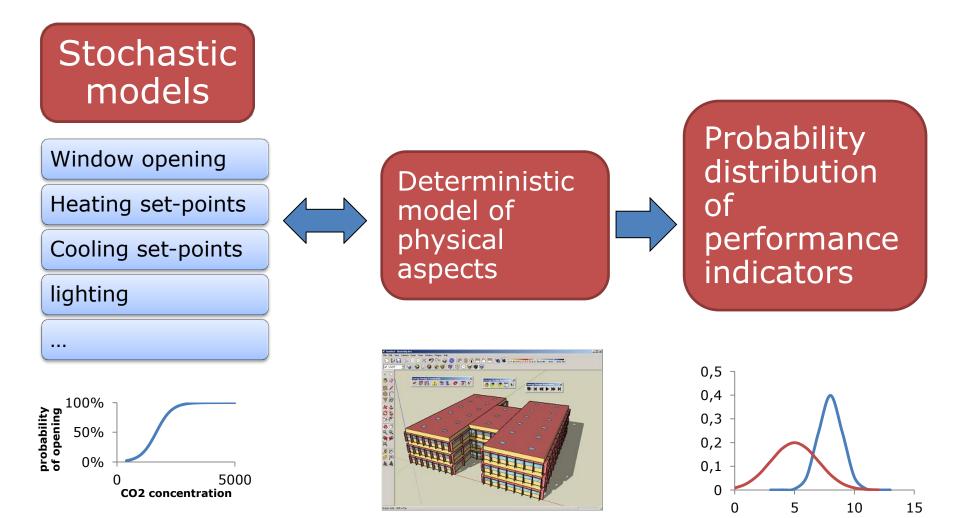
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Shared or individual heating cost



From deterministic to stochastic modelling



Energy Efficient Technologies

- Indoor air quality
 - Reduce loads (pollution sources)
 - Heat recovery
 - Increase system efficiency
 - Natural ventilation-Hybrid ventilation
 - Air distribution (contaminant removal) effectiveness
 - Personal ventilation
 - Air cleaning
- Thermal comfort
 - Reduce loads (building shell, solar screen, internal loads)
 - Increase system efficiency
 - Low Temperatur Heating- and High Temperature Cooling Systems
 - Use of building mass to reduce peaks (Thermo-Active-Building-Systems (TABS))
 - Drifting indoor temperatures



